Simultaneous incorporation of mass and force terms in the multi-relaxation-time framework for lattice Boltzmann schemes

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This paper presents an analysis of the simultaneous incorporation of force and mass source terms into the multi-relaxation-time (MRT) collision operator. MRT force incorporation was obtained through Chapman–Enskog analysis. The numerical scheme was tested on different benchmark problems, including the decay of a shear wave with different bulk and kinematic viscosities and axisymmetric flow.

Keywords: lattice Boltzmann equation; multi-relaxation times; force incorporation; sound wave decay

1. Introduction

The lattice Boltzmann equation (LBE) is a new technique for solving fluid dynamics and transport problems. Unlike conventional computational fluid dynamics, the LBE is based on the kinetic theory. The kinetic nature brings LBE many advantages and enables it to be an appealing tool for modelling and simulating fluid flows. In the past two decades, LBE has been successfully applied for a variety of hydrodynamics problems [1]. In many fluid flows, the working fluid is usually exposed to an external force field (e.g. gravity) or internal interactions (e.g. van der Waals forces). In order to capture the physics in such systems, it is critical to treat the force correctly in the LBE framework.

It is well understood that LBE models can be classified into two main types in terms of the collision operator employed, i.e. single-relaxation-time (SRT) or Bhatnagar–Gross–Krook (BGK) model and multi-relaxation-time (MRT) model. BGK–LBE models are widely used and have gained much success in solving a variety of fluid flow problems. On the other hand, MRT–LBE models have attracted much attention recently owing to some noticeable features, such as enhanced numerical stability, exact realization of boundary conditions, etc. [2].

There have been successful implementations of a body force in the literature for lattice BGK models [3–5]. However, MRT force implementation has not yet been addressed thoroughly, except for a few publications [6–8]. Furthermore,

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most of the previous studies on force implementation, for either BGK or MRT–LBE models, did not consider the cases when a source exists in the continuity equation. The first related work on this subject is due to Halliday et al. [9], who treated the axisymmetric Navier–Stokes equations in cylindrical coordinates as two-dimensional equations with mass and force terms in pseudo-Cartesian coordinates, which was solved by a lattice BGK model with a forcing term that can replicate the sources in the continuity and momentum equations. Some other versions were also developed later with similar ideas. Another work considering both mass source and body force in LBE is attributed to Ginzburg et al. [8], which is developed within the framework of a two-relaxation-time (TRT) model. To the best of the authors’ knowledge, there are no reports so far on implementation of a forcing term accounting for both mass source and body force in the full MRT–LBE.

In this work, we present an analysis of the MRT–LBE with a forcing term that can recover the mass source and body force through Chapman–Enskog expansion. The expressions of the forcing term in moment space are found explicitly. It is found that the force implementation moments depend on the eigenvalues of the collision operator. When all the eigenvalues coincide, then the MRT implementation agrees with that of Guo et al. [4]. Some numerical tests, including axisymmetric flow in a concentric pipe, and a decaying wave with different bulk and shear viscosities, are then carried out to validate the proposed algorithm.

The paper is organized as follows. We briefly mention the lattice Boltzmann equation with the force term in §2. The Chapman–Enskog analysis is performed in §3 for the most spread matrix for the two-dimensional D2Q9 model. In §4, the results are compared with analytical solutions for axisymmetric flow and wave decay with different bulk and kinematic viscosities.

## 2. Lattice Boltzmann method

The MRT lattice Boltzmann approach is as follows:

\[
m_i = M_{ij} f_j, \quad m_i^* = m_i - s_i (m_i - m_i^{eq}) + m_{iF}, \quad f_i (x + c_i, t + 1) = M_{ij}^{-1} m_j^*. \tag{2.1}
\]

This can be rewritten for the sake of clarity and simplicity in the matrix model with normalization factors and relaxation times as follows:

\[
\begin{align*}
    f_i^* &= f_i (x, t) - M_{ij} f_j (x, t) - f_j^{eq} (x, t) + F_i \\
\end{align*}
\]

and

\[
    f_i (x + c_i, t + 1) = f_i^*. \tag{2.2}
\]

Here the collision matrix \( M_{ij} \) consists of the normalized eigenvectors and relaxation times:

\[
M_{ij} = \sum_k M_i^k \frac{\omega_k}{|M_k^i|} M_j^k, \tag{2.3}
\]

and \( F_i \) is the force population responsible for the force inclusion. In this work, we took the two-dimensional nine-velocity (D2Q9) model as an example, where the
transform matrix $M$ can be found in the work by Lallemand & Luo [2]. The moments related to the matrix are expressed as
\[
m = M f = (\rho, \epsilon, \epsilon, j_x, q_x, j_y, q_y, p_{xx}, p_{xy}),
\]
which are known for the equilibrium distribution function as $m^{eq} = (\rho, -2\rho + 3\rho (u_x^2 + u_y^2), \rho - 3\rho u_x, \rho u_y, -\rho u_x, \rho (u_x^2 - u_y^2), \rho u_x u_y)$. The algorithm consists of two processes—collision and propagation. The collision is performed in moment space. The propagation is done in the distribution population space. Note that, in comparison with the one relaxation parameter for BGK, the MRT approach operates with nine collision rates corresponding to moments $u_r, u_e, u_3, u_{jx}, u_{qx}, u_{jy}, u_{qy}, u_{pxx}$ and $u_{pxy}$. If all the relaxation rates equal each other, then the MRT–LBE is equivalent to the most spread BGK–LBE.

Matrix $M_{ij}$ has certain features. The most used feature in this work is when the vector $M_l$ multiplied on matrix $M_{ij}$ and summed up by index $i$ produces the eigenvector multiplied on the corresponding eigenvalue:
\[
\sum_i M_j^l M_{ij} = \sum_{ik} \frac{\omega_k}{|M_k|} M_{ji}^k \sum_i \frac{\omega_k}{|M_k|} M_{ji}^k = \omega_i M_j^l.
\]

3. The Chapman–Enskog analysis

In the following paragraphs, the equilibrium distribution function and the force populations $F_i$ will be derived to restore the Navier–Stokes equation with the mass and force terms:
\[
\begin{aligned}
\partial_t \rho + \partial_a \rho u_a &= S \\
\partial_t \rho u_a + \partial_a \rho u_a u_a &= F_a - \partial_a p + \eta \partial_{\beta} \rho \partial_{\beta} u_a + \zeta \partial_{\beta} \rho \partial_{\alpha} u_a.
\end{aligned}
\]

The moments for the D2Q9 model (2.4) imply the macroscopic quantities as $\rho = \sum_i M_i^0 f_i = \sum_i f_i$ and $\rho u = \sum_i M_i^j f_i = \sum_i f_i c_i$. However, in the presence of the source terms, the macroscopic parameters are not changed [4,8] in a unique way. One possible choice is to take macroscopic parameters with half shifted source terms [8]:
\[
\begin{aligned}
\rho^m &= \rho^{eq} = \rho + \epsilon \frac{S}{2} = \sum_i f_i + \epsilon \frac{S}{2} \\
\rho^m u^m &= \rho^{eq} u^{eq} = \sum_i f_i c_i + \epsilon \frac{F}{2},
\end{aligned}
\]
where $\epsilon$ is the Knudsen parameter, and index $m$ stands for the macroscopic parameters. The Chapman–Enskog analysis is done with the expansion of the distribution function in terms of the Knudsen number as $f_i = \sum_{n=0}^{\infty} \epsilon^n f_i^{(n)} = f_i^{eq} + \sum_{n=1}^{\infty} \epsilon^n f_i^{(n)}$. The time derivative is expanded as $\partial_t = \partial_0 + \epsilon \partial_1 + \cdots$. From system (3.2) one can obtain the moments for $f_i^{(1)}$ (note that higher-order terms with $n > 2$ are eliminated for the conservation laws to be fulfilled):
\[
\begin{aligned}
\sum_i f_i^{(1)} &= -\frac{S}{2} \\
\sum_i f_i^{(1)} c_i &= -\frac{F}{2}.
\end{aligned}
\]
After the substitution of the series expanded distribution function into the LBE (2.2), the infinite consecutive series of equations (Chapman–Enskog system) can be obtained:

\[ e^0: \quad f_i^{(0)} = f_i^{eq}, \]

\[ e^1: \quad (\partial_t + c_i \partial_x) f_i^{(0)} = - \sum_j M_{ij} f_j^{(1)} + F_i - \sum_j M_{ij} f_j^{(2)} \]

and

\[ e^2: \quad \partial_t f_i^{(0)} + (\partial_t + c_i \partial_x) \left( f_i^{(1)} - \frac{M_{ij} f_j^{(1)}}{2} + \frac{F_i}{2} \right) = - \sum_j M_{ij} f_j^{(2)} . \]

By multiplying equations (3.4) with the eigenvector \( M^\rho \), one can restore the continuity equation:

\[ e^1: \quad \partial_t \rho^m + \partial_x \rho u^m = - \omega \sum_j M^\rho_{ij} f_j^{(1)} + \sum_i M^\rho_i F_i \]

and

\[ e^2: \quad \partial_t \rho^m + \partial_x \left( \sum_i f_i^{(1)} - \omega \sum_j M^\rho_{ij} f_j^{(1)} \right) + \partial_x \sum_i F_i M^\rho_i = 0. \]

After substitution of the known moments of \( f_i^{(1)} \) (3.3) and specification of some of the force moments (\( \sum_i F_i = S(1 - \omega / 2) \) and \( \sum_i F_i c_{ia} = F_a(1 - \omega_{ja} / 2) \)), system (3.5) is considerably simplified:

\[ \partial_t \rho^m + \partial_x \rho u^m = S \quad \text{and} \quad \partial_t \rho^m = 0, \]

which restores the continuity equation by taking into account that \( \partial_t = \partial_t + \epsilon \partial_n \). The next iteration, the Chapman–Enskog system summed up with \( M^\lambda \), gives:

\[ e^1: \quad \partial_t \rho^m u^m + \partial_x \sum_i c_{i\beta} c_{ia} f_i^{(0)} = \frac{\omega_{ja} F_a}{2} + F_a \left( 1 - \frac{\omega_{ja}}{2} \right) = F_a \]

and

\[ e^2: \quad \partial_t \rho^m u^m + \partial_x \left( \sum_i c_{i\alpha} c_{i\beta} f_i^{(1)} - \frac{\sum_i c_{i\alpha} c_{i\beta} M_{ij} f_j^{(1)}}{2} \right) + \partial_x \sum_i c_{i\alpha} c_{i\beta} F_i = 0. \]

where already known force moments are substituted and used. We require terms containing \( c_{i\alpha} c_{i\beta} \) to be expressed through the eigenvectors of matrix \( M \) to perform the summation \( \sum_i c_{i\alpha} c_{i\beta} M_{ij} \):

\[ c_{ix} c_{ix} = \frac{M_i^e + 4 M_i^\rho + 3 M_i^{pex}}{6}, \quad c_{iy} c_{iy} = \frac{M_i^e + 4 M_i^\rho - 3 M_i^{pex}}{6} \quad \text{and} \quad c_{ix} c_{iy} = M_i^{py}. \]
Therefore, the term \( \sum_i c_{i\alpha} c_{i\beta} f_i^{(1)} - \sum_i c_{i\alpha} c_{i\beta} M_{ij} f_j^{(1)} / 2 \) can be represented through the sum of terms that have the form \( (1 - \omega_{c,p\alpha}/2) \sum_i M_{i,j}^{c,p\alpha} f_i^{(1)} \). The moments \( M^X f_i^{(1)} \) in the expression above can be found from the summation of the second equation of the Chapman–Enskog system (3.4) multiplied with the corresponding eigenvectors \( M^X \):

\[
\partial_t \sum_i M^X_i f_i^{(0)} + \partial_j \sum_i M^X_i c_{i\beta} f_i^{(0)} = -\sum_j M^X_i M_{ij} f_j^{(1)} + \sum_i M^X_i F_i. \tag{3.9}
\]

After lengthy algebra using the decomposition of the terms as \( M^X_i c_{i\alpha} \) through the matrix eigenvectors, the moments of \( f_i^{(1)} \) can be obtained:

\[
\begin{align*}
\sum_i M^e_i f_i^{(1)} &= \frac{1}{\omega_e} \left( \sum_i M^e_i F_i - \partial_t \left( -2\rho^m + 3\rho^m (u_x^m + u_y^m) \right) \right), \\
\sum_i M^{pxx}_i f_i^{(1)} &= \frac{1}{\omega_{pxx}} \left( \sum_i M^{pxx}_i F_i - \partial_t \rho^m (u_x^m - u_y^m) \\
&\quad - \frac{2}{3} \partial_x \rho^m u_x^m + \frac{2}{3} \partial_y \rho^m u_y^m \right), \\
\text{and} \quad \sum_i M^{pyy}_i f_i^{(1)} &= \frac{1}{\omega_{pyy}} \left( \sum_i M^{pyy}_i F_i - \partial_t \rho^m u_x^m u_y^m - \frac{1}{3} \partial_x \rho u_x^m - \frac{1}{3} \partial_y \rho u_y^m \right). \tag{3.10}
\end{align*}
\]

The expressions for \( M^e_i \) and \( M^{pxx}_i \) contain partial derivatives \( \partial_t \rho^m \) and \( \partial_t \rho^m u_x^m u_y^m \). These can be obtained through the continuity equations:

\[
\partial_t \rho^m = -\partial_j \rho^m u_j^m + S \quad \text{and} \quad \partial_t \rho^m u_{\alpha} = -\frac{1}{3} \partial_{\alpha} \rho^m - \partial_j \rho^m u_{\alpha} u_j^m + F_\alpha. \tag{3.11}
\]

The third-order terms such as \( u_{\alpha}^m u_{\beta}^m \partial_j \rho u_j^m \) and \( u_{\alpha}^m u_{\beta}^m S \) can be neglected:

\[
\partial_t \rho^m u_{\alpha}^m u_{\beta}^m \approx u_{\alpha}^m \left( -\frac{1}{3} \partial_{\beta} \rho^m + F_\beta \right) + u_{\beta}^m \left( -\frac{1}{3} \partial_{\alpha} \rho^m + F_\alpha \right). \tag{3.12}
\]

After the moments, calculations of \( f_i^{(1)} \) (equation (3.10)) and substitution into the system (3.7), the force moments should have the form

\[
\begin{align*}
\sum_i M^e_i F_i &= \left( 1 - \frac{\omega_e}{2} \right) \left( 6 u_x^m F_x - 2 S \right) \\
\sum_i M^{pxx}_i F_i &= \left( 1 - \frac{\omega_{pxx}}{2} \right) \left( 2 u_x^m F_x - 2 u_y^m F_y \right), \\
\text{and} \quad \sum_i M^{pyy}_i F_i &= \left( 1 - \frac{\omega_{pyy}}{2} \right) \left( u_x^m F_y + u_y^m F_x \right). \tag{3.13}
\end{align*}
\]
in order to restore the full system of Navier–Stokes equation with a force and the continuity equation with a mass source term (3.1), where \( \omega_\eta = \omega_{pzz} = \omega_{pzy} \), \( \omega_\zeta = \omega_z \) and the viscosities are \( \eta, \zeta = \frac{1}{3}(1 - \omega_\eta \zeta/2) \). When all the eigenvalues coincide with each other, the force representation coincides with the Guo force term representation [4].

4. Numerical results

(a) Flow in a concentric annular pipe

A flow in a concentric annular pipe driven by a constant pressure gradient is simulated. The radius of the inner and outer cylinders are \( r_1 \) and \( r_2 \), respectively. With the transformation \( (z, r) \to (x, y) \) and \( (v_z, v_r) \to (u, v) \), the governing equations of the flow can be expressed in pseudo-Cartesian coordinates as [9]:

\[
\begin{align*}
\partial_x u + \partial_y v &= S, \\
\rho D_t u &= -\partial_x p + v \nabla^2 u + F_x \\
\rho D_t v &= -\partial_y p + v \nabla^2 v + F_y,
\end{align*}
\]

where \( S = \rho - v/y \), \( F_x = (\rho v/y) \partial_y u \) and \( F_y = (\rho v/y) (\partial_y v - v/y) \); \( \nabla^2 = \partial_x^2 + \partial_y^2 \) is the Laplacian operator in the pseudo-Cartesian coordinates. The analytical solution of this flow at steady state is

\[
u(y) = \frac{Gr^2}{4y} \left[ 1 - \left( \frac{y}{r_1} \right)^2 - \frac{1 - \alpha^2}{\ln \alpha} \ln \left( \frac{y}{r_1} \right) \right], \tag{4.2}
\]

where \( G = -\partial_x p/\rho \) is a constant, and \( \alpha = r_2/r_1 \) is the radius ratio.

In the test, we set \( r_1 = 100 \) and \( r_2 = 132 \) or 164, and the length of the pipe is set to 8. The driven force is \( 10^{-5} \), and periodic boundary conditions are applied to the inlet and outlet of the pipe, and the half-way bounce-back scheme is applied to the surfaces of the inner and outer cylinders. The gradients in \( F_x \) and \( F_y \) are discretized using the second-order central finite-difference scheme. In figure 1, the velocity profiles predicted by the MRT–LBE with the present force formulation are compared with the analytical solutions at different values of \( \omega_\eta \) with \( \omega_\zeta = 0.8 \). Good agreement between the numerical results and the analytical solutions is observed, which confirms the validity of the force scheme.

(b) Wave decay with different bulk and shear viscosities

The system in the region of the one-dimensional sound waves with the mass term in the advection–diffusion equation can be represented in terms of pressure \( p = c_s^2 \rho \) and the flux \( j_x = \rho u_x \):

\[
\partial_t p + c_s^2 \partial_x j_x = c_s^2 S \quad \text{and} \quad \partial_t j_x = -\partial_x p + F_x + (\eta + \zeta) \partial_x^2 j_x. \tag{4.3}
\]

For simplification, the initial conditions along with the mass and force terms are assumed as

\[
\begin{align*}
p(x, t = 0) &= 0, \\
j_x(x, t = 0) &= A \cos(kx), \\
F_x(x, t) &= B \cos(kx), \\
S(x, t) &= C \cos(kx). \tag{4.4}
\end{align*}
\]
The overall solution satisfying the initial conditions (4.3) can be written as

\[
p(x, t) = \sin(kx) \left[ e^{\lambda_{\text{real}} t} \left( \sin(\lambda_{\text{imag}} t) \left( \frac{A c_s^2 k}{\lambda_{\text{imag}}} + \frac{B \lambda_{\text{real}}}{k \lambda_{\text{imag}}} \right) - \frac{B \cos(\lambda_{\text{imag}} t)}{k} \right) + \frac{B}{k} \right] \\
+ C \cos(kx) \left[ e^{\lambda_{\text{real}} t} \left( \sin(\lambda_{\text{imag}} t) \left( \frac{c_s^2}{\lambda_{\text{imag}}} + \frac{\lambda_{\text{real}} (\eta + \zeta)}{\lambda_{\text{imag}}} \right) \right) + (\eta - \zeta) \cos(\lambda_{\text{imag}} t) \right] + \eta + \zeta
\]

and

\[
\begin{align*}
\lambda_{\text{real}} &= -k^2(\eta + \zeta)/2 \\
\lambda_{\text{imag}} &= k\sqrt{4c_s^2 - k^2(\eta + \zeta)^2}/2
\end{align*}
\]

where \( \lambda_{\text{real}} = -k^2(\eta + \zeta)/2 \) and \( \lambda_{\text{imag}} = k\sqrt{4c_s^2 - k^2(\eta + \zeta)^2}/2 \). The numerical simulation is performed on a domain of 40 \times 1 with the parameters \( A = 0.001, B = 0.002 \) and \( C = 0.003 \). A few examples for \( \omega = 1.0 \) and \( \omega_{\text{bulk}} = 0.8, 1.2 \) along with the analytical profiles are given in figure 2. We examined the range of parameters \( \omega \) and \( \omega_{\text{bulk}} \) with the given force and mass source representations. The relative error is introduced as \( e = |j_{x, \text{analytical}} - j_{x, \text{simulation}}|/j_{x, \text{amp}} \), where \( j_{x, \text{amp}} \) is the amplitude of the signal that bounds the oscillating solution. One of the possible measurements of the error can be a least-squares fit of the signal and simulation to calculate \( \lambda_{\text{real}} \) and \( \lambda_{\text{imag}} \). The results are shown in figure 3.
Figure 2. Examples of the signal for $\omega = 1.0$ and (a) $\omega_{\text{bulk}} = 0.8$ and (b) $\omega_{\text{bulk}} = 1.2$ along with the analytical profiles for $x = 0$. Solid line, analytical; plus symbols, simulation. (Online version in colour.)

Figure 3. Relative error for different $\omega$ and $\omega_{\text{bulk}}$ for $x = 0$ and for $t = 0, \ldots, 100$. Dashed line with plus symbols, $\omega_{\text{bulk}} = 0.6$; thick line with filled circles, $\omega_{\text{bulk}} = 1.0$; dotted line with triangles, $\omega_{\text{bulk}} = 1.4$. (Online version in colour.)

5. Conclusion

This paper demonstrated the simultaneous incorporation of the mass and force terms in the frame of the MRT collision operator. A few benchmark problems are solved, and predicted results were compared with analytical solutions. Results show that the approach can be used for the simulation of fluid flow with source terms. However, more tests need to be done. For example, the results are useful for researchers working in the LBE combustion and reacting flows.

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